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U.S. National Phase Application Serial No. 10/598,528

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1 SAFETY DEVICE

2 CROSS-REFERENCE TO RELATED APPLICATION

3 The Applicants claim the benefit, under 35 U.S.C. § 119(a)-(d), 365(a), and/or 365 (b) of
4 PCT International Application No. PCT/DE 2005/000388, filed March 3, 2005, entitled
5 “SAFETY UNIT,” which claims the benefit of German Patent Application No. 10 2004 010
6 987.7 filed March 3, 2004. The entire content of the PCT International Application No.
7 PCT/DE2005/000388 is incorporated herein by this reference.

8
9 TECHNICAL FIELD OF THE INVENTION

10 The present invention relates to a safety device for detachable clamping of elements that
11 move relative to each other.

12
13 BACKGROUND OF THE INVENTION

14 In machine construction, for example, in an assembly installation, linear guidance
15 systems are known that work in the vertical direction. In the process, the components to be
16 moved, which may be very heavy, are moved up and down preferably in the vertical direction.
17 To secure the moved loads, for example, in the case of a system failure with pressurized air
18 stoppage or electrical power outage, clamping systems are known in practice that are capable of
19 clamping the load to be carried in a substantially arbitrary vertical position relative to a guide, so
20 that the load is not lowered or does not fall due to gravity.

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1 In particular, clamping systems are known in which a clamp element is provided with a
2 chamber into which a medium can be admitted, which chamber, during the admission, undergoes
3 changes in its geometric dimensions such that, as a result, a force is applied or released against a
4 stationary component relative to the moved load, to allow a clamping or release of the relative
5 movement between the load and the component. This stationary component can be particularly
6 the vertical guide. Usually pressurized air or hydraulic oil is admitted into the chamber.

7 In particular, there is a known design of a clamping system that, in the pressureless state
8 (when there is no pressure in it), generates a clamping force, which design as a result of the
9 pressure admission caused by a corresponding deformation of the chamber releases the clamping
10 force and allows the relative movement between the weight-bearing part and the fixed
11 component. This system presents the advantage that in the case of a stoppage of the pressurized
12 air or the corresponding hydraulic medium, the clamping force is applied, and the relative
13 movement is stopped, preventing further movement of the weight in the case of a system failure.

14 However, the disadvantage of the designs known from the state of the art is that,
15 following a failure as described above, the connection of the corresponding medium results in
16 the clamping force being released again, and the weight being again set in motion as a result of
17 gravity. This can cause considerable harm, both to equipment and to humans, if the connection
18 of the medium occurs without sufficient consideration or safety provisions for the possible
19 subsequent movement of the weight. The safety mechanism thus works only to the extent that a
20 failure of the system pressure stops a relative movement. The consequences of reestablishing the

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1 system pressure are the responsibility of only the operating personnel and are thus associated
2 with risk.

3 The problem of the invention therefore is to provide a safety device that allows a reliable
4 new startup of the above-mentioned installations, and that has a construction that is simple and
5 cost-effective to carry out.

7 SUMMARY OF THE INVENTION

8 Provided is a safety device with a clamping system that uses pressure admission into a
9 chamber of a clamp element dependent on the position of the weight to be carried relative to the
10 clamping system by using an appropriate construction design. For this purpose, a valve is
11 provided in a medium supply line, which valve is to be opened or closed by means of a movable
12 actuation element. Here, the actuation element should be movable in the same direction in
13 which the weight to be moved moves, that is, preferably in a vertical direction Z.

14 As a result of the arrangement of such a valve, it becomes possible to couple the vertical
15 movement or position of the weight with an actuation of the valve, and thus to provide an
16 additional control for the pressure admission into the chamber or the clamping force resulting
17 therefrom.

18 Below, the term "opening" of the valve denotes a movement of the actuation element
19 such that the supply into the chamber of the clamp element undergoes a pressure release, thus
20 preventing a pressure buildup in the chamber. Conversely, the term "closing" below means that

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1 the medium fed into the system cannot escape, and thus pressure is admitted with force into the
2 chamber of the clamp element.

3 In the simplest embodiment of the invention, a base body is provided, which can be
4 traversed by a component extending in an axial direction in such a manner that the base body
5 and the component represent two elements that are moved relative to each other. According to
6 the state of the art, the component is considered here to be a fixed component, for example, in
7 the shape of a vertically-oriented rod as a guide element. The base body slides up and down
8 along this component.

9 The base body presents at least one clamp element, which contains a chamber into which
10 a medium can be admitted through a medium supply. On or to the chamber, a force transmission
11 element is coupled, which is designed for transmitting a clamping force from the chamber to the
12 component. The component, which is preferably arranged vertically, receives the force in the
13 radial direction, that is, substantially in the horizontal direction if the component is oriented
14 vertically.

15 The term "radial" below refers to a direction toward the component. It is preferred for
16 this direction to extend precisely perpendicularly with respect to a longitudinal direction of the
17 component; however, other angles between the longitudinal axis of the component and the
18 "radial" direction besides 90° are also included. The longitudinal axis of the component in this
19 connection describes the direction of movement of the base body or of the clamp element
20 relative to the component in the unclamped state.

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1 It is also conceivable here for the longitudinal axis of the component to be curved, that is,
2 to use a guide that is not necessarily shaped exclusively rectilinearly, for example, in the case of
3 a component in the shape of a curved, or even circular, rail.

4 Because the cross section of the component perpendicular with respect to its axis can be
5 not only circular, but also, for example, rectangular or in any desired other profile shape of a
6 guide element, the expression "radial" therefore denotes not only a circular element. Rather,
7 "radial" also describes the circumstance where a clamping force can be applied from several
8 sides onto components that are arranged centrally with respect to it.

9 The clamp element, during the admission of medium into the chamber, is deformed
10 elastically and thus undergoes a change in its dimensions. In particular, the dimension in the
11 radial direction changes, so that, as a result, the component receives a force or force is removed
12 from it via the force transmission element, in the radial direction relative to the base body, and
13 consequently it is clamped or released.

14 Some embodiments also allow a permanent clamping (or also release) independent of
15 whether the system pressure is built up again after a system breakdown.

16 In an advantageous embodiment of the invention, the actuation element is arranged on a
17 weight body, or on a part thereof, which weight body is guided in the Z direction. By means of
18 this simplification of the construction, a direct connection is established between the vertical
19 position of the weight and the opening or closing of the valve. Thus, a movement of the weight
20 in the Z direction opens or closes the valve. By an appropriate selection of the clamp elements

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1 (for example, such that a clamping force is applied when the chambers have no pressure), one
2 can advantageously achieve that a clamping force is never applied if the weight body has opened
3 the valve as a result of its movement and thus released the pressure from the medium supply line
4 so that it is pressureless.

5 If the base body with a clamping system formed therein is coupled with the weight to be
6 transported in an appropriate manner, then it receives the force of the weight. For this purpose,
7 the base body can be arranged, for example, adjacent to and above the weight body to be
8 transported.

9 The valve, which can be formed particularly in the base body itself, is closed during the
10 operation by the closely abutting weight body, so that pressure is applied to the clamping system
11 and the clamping system allows a relative movement between the base body with the attached
12 weight body, on the one hand, and the guide component fixed relative to the weight body, on the
13 other hand.

14 A pressure drop in the system then results in a pressure release from the originally
15 pressurized chamber in the clamping system, and consequently a clamping force is generated
16 that should prevent additional movement of the base body together with the attached weight
17 body. While the base body with the clamping system located therein is retained in its vertical
18 position at the time the clamping force is generated, the weight body attached to the base body
19 continues to be subjected to the gravitational force and thus separates from the base body located

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1 above it. As a result, the valve of the medium supply, which is already no longer under pressure,
2 is opened.

3 An additional downward movement of the weight body relative to the base body should
4 be prevented by the above-mentioned coupling. It is conceivable here to use a simple guide rod
5 system, which, while allowing the movement by shifting of the weight body to a selectable
6 extent relative to the base body, maintains the weight body in the vertical or Z direction at a
7 predetermined separation from the base body by means of appropriate coupling to this base
8 body.

9 After system failure, the base body with the blocking clamping system is thus in a given
10 vertical position, while the weight body is suspended, and also motionless, with a separation,
11 which is preferably chosen to be small, under the base body. The separation between the base
12 body and the weight body is advantageously chosen in such a manner that it ensures a sufficient
13 opening of the valve, which occurs for a separation of a few millimeters.

14 If the system pressure is then built up again, then the medium supply indeed admits the
15 corresponding medium; however, because the valve in the base body is opened as a result of the
16 weight body no longer abutting, the chamber of the clamp element in the base body can no
17 longer be pressurized and therefore cannot release the clamping. Any shifting or an unintended
18 lowering of the base body together with the weight body is thus for the time being ruled out.

19 The clamping of the base body, even after the system pressure has been reestablished,
20 can only be released if the weight body is moved upward again against the base body in such a

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1 manner that the valve is closed and thus the pressure in the clamping system is built up again.
2 For example, if the weight body is driven or moved by means of an axle that is parallel to the Z
3 direction (for example, by means of a lifting unit or a lifting system), and the fixed component
4 functions only as a guide or safety element, then it is necessarily ensured that the weight body
5 (together with the base body and the clamping system arranged above it) becomes freely
6 movable again only when the axle or lifting unit arranged parallel to the weight body and
7 belonging to the same pressure system works again as intended and can lift and carry the weight
8 body.

9 In another embodiment of the invention, the coupling between the base body and the
10 weight body is provided such that the weight body is pressed by a spring force in the Z direction
11 against the base body. Such a spring force ensures that, when the clamping system is released, no
12 relative movement occurs between the base body and the weight body. Rather, these two bodies
13 are in principle applied against each other, so that the valve is closed and the pressure admission
14 into the chamber in the clamping system is ensured. The spring force here should be chosen
15 such that in the case of a pressure drop and the resulting clamping of the base body, the weight
16 force of the weight body exceeds the spring force to such an extent that the weight body
17 separates in the vertical direction downward away from the base body, thus opening the valve.

18 In an additional advantageous embodiment, the valve is designed such that, in the opened
19 position, it allows the medium to flow from the medium supply line into the environment. This
20 is particularly advantageous when compressed air is admitted because the construction costs are

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1 then very low. By simply opening the supply line to the environment during the opening of the
2 valve, the pressure in the supply line and thus also in the chamber of the clamping system is
3 released.

4 In a similar embodiment, on the other hand, the valve is designed so that the medium is
5 recycled in a medium recycling line provided particularly for that purpose, when the valve is
6 opened. This can be required particularly if the chamber of the clamping system is operated with
7 hydraulic oil or a similar fluid. Because the hydraulic oil cannot be released into the
8 environment, recycling of the hydraulic oil through an appropriate line is made possible. Again,
9 it is essential for the functioning of this valve that the valve, in the opened state, prevent the
10 pressurization of the chamber of the clamping system, so that the pressure in the chamber is
11 released, thus allowing a clamping effect to be built up.

12 An additional advantageous embodiment of the invention provides for one or more clamp
13 elements to be designed so that they surround the substantially fixed component (rod, guide) in
14 the form of an annular membrane. Such a membrane, which can be designed particularly to be
15 relatively flat, is constructed to generate high radial forces with, relative to them, a small change
16 in the dimensions in the Z direction. The forces here are determined substantially on the basis of
17 a rhombus, where the near and opposite tips are exposed to opposite forces, so that the result is a
18 higher force at the other opposite tips that are farther from each other, using a small setting path.

19 The clamp element and/or the chamber can consist of any appropriate material, where it
20 is advantageous for the chamber walls to be constructed so that they are resistant to tension or

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1 pressure, to be able to transmit a particularly strong force. In particular, they may be made
2 entirely or partially of metal.

3 Instead of designing the component or rod, respectively, as a preferably fixed element, an
4 additional advantageous embodiment of the invention provides for designing the component or
5 rod as a movable element, which is intended to be moved in space (preferably up and down). On
6 the other hand, the actuation element described above as substantially movable, should now be a
7 fixed, or part of a fixed, component. The described base body with the clamping system formed
8 therein here acts again in the above-described manner together with the actuation element, which
9 is arranged in a fixed manner, or within a fixed component. By such an arrangement, the
10 described principle continues to be applied; however, in spite of the replacement of the formerly
11 fixed rod with a now moved component, and the replacement of the substantially co-moved
12 actuation element with a now substantially fixed actuation element, the collaborative functioning
13 of these components is maintained.

14 In a simple embodiment according to this principle, the base body which at least one
15 clamp element thus, as before, acts in the above-described manner together with the component
16 that traverses this base body in the axial direction, where, in this embodiment, the component
17 (rod or similar part) performs the vertical movement proper, while the base body if need be
18 experiences a slight deflection, as shown below. The supply of medium for the clamp element is
19 also carried out in accordance with the above-mentioned embodiments through the base body,
20 where again a valve formed between the base body and the actuation element can also influence

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1 the supply or reduce the supply pressure. However, in this process, the actuation does not occur
2 due to the movement of the (now substantially fixed) actuation element, but due to the
3 movement of the base body, which is arranged immediately below this fixed component, relative
4 to the actuation element.

5 The base body here can be shifted to a desirable extent relative to the fixed component in
6 the axial direction, so that according to the principle that is the foundation of the invention, the
7 supply of medium to the base body undergoes a pressure reduction, and thus becomes without
8 effect, if the base body is separated from the fixed component to an extent, thus opening a slit
9 through which the medium can escape.

10 Alternatively, a medium supply is also conceivable through the fixed component into the
11 base body, which is interrupted by opening the valve (formation of a slit between the fixed
12 component and the base body). The following is a brief explanation of the operating principle:

13 During normal operation, the connection of the medium supply to the base body from
14 below is applied directly against the fixed component, through which the medium supply is
15 made available. The clamp element(s) is (are) preferably pressurized, so that no clamping effect
16 is built up against the rod, guide, etc., which is moved axially with respect to the base body.
17 Thus, the rod is freely movable in the axial direction, so that weights attached to or arranged on
18 it can be co-moved up or down.

19 In the case of a pressure stoppage, the pressure of at least one clamp element is reduced
20 so that a clamping force is generated between the base body and the vertically moved rod. The

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1 rod, optionally with weights arranged on it, is then pulled down by gravity, where the base body
2 that is firmly clamped to the rod now also undergoes this movement.

3 However, an abutment provided between the component, which is fixed and functions as
4 actuation element, and the base body catches the base body and the rod clamped to it in such a
5 manner that the rod cannot be moved farther down over the abutment. Thus, the rod is reliably
6 protected from an uncontrolled and possibly dangerous lowering.

7 In the case of a new startup of operation and the buildup of pressure in the medium
8 supply, this pressure can at first not be transmitted up to the clamp element of the base body,
9 because, as a result of the lowering of the base body with the rod, a slit has formed between the
10 fixed component and the base body, which releases the introduced medium into the environment
11 and which does not reach the base body. The rod thus remains stopped, in spite of the new
12 availability of medium, in its downward position. (The same effect naturally also occurs
13 according to the invention if – as described above – the medium supply reaches directly into the
14 base body and presents only one opening in the direction of the fixed component arranged above
15 it.)

16 However, if the rod (together with the base body clamped to it) is lifted in a targeted or
17 intentional manner, so that the slit between the base body and the fixed component is closed,
18 then the medium penetrates into the clamp element(s). As a result, the base body is separated
19 from the vertically moved rod so that the latter can later be moved up and down in a targeted
20 manner.

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1 To ensure that the base body maintains its (abutting) position relative to the fixed
2 component acting as the actuation element, an appropriate spring mechanism can be provided,
3 which presses the base body with sufficient spring force from below against the fixed
4 component, thus ensuring the supply of medium, or the maintenance of the medium pressure,
5 into the clamp elements. It is advantageous to design the spring mechanism here in such a
6 manner that the base body moves downward away from the fixed component only if an
7 additional force (particularly the gravitation force of the rod in the case of the clamped state)
8 also acts on the base body, pulling it down.

9 The above-described device can be used particularly in the case where the vertical
10 component is to be moved, while the remaining components (base body or fixed component
11 functioning as actuation element) should be substantially or completely fixed in space. All the
12 dependent characteristics, such as the constitution of the base body or the shape and position of
13 the rod, are here intended to be valid for all embodiments of the present invention.

14 In particular, for this reversal of function, it is conceivable to design an actuation element
15 that does not release the medium freely into the environment, but into a selected medium
16 recycling line, as would be required particularly for hydraulic oil. In this case, the actuation
17 element would be attached in the fixed component in such a manner that, as a result of a
18 downward movement of the base body, it opens an outlet for the medium to flow into the
19 medium recycling line. If the base body moves in the above-described manner repeatedly
20 upward to the desired extent, then the actuation element closes off the path into the medium

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1 recycling line, so that the pressure made available by the medium supply can again act on the
2 clamp elements to release the clamping.

3 Additional advantageous embodiments are within the scope of the disclosure herein.
4

5 **BRIEF DESCRIPTION OF THE DRAWINGS**

6 Figure 1 shows a cross sectional view of a first embodiment of the invention in a position
7 with a closed valve.

8 Figure 2 shows the same embodiment as in Figure 1 in a position with an opened valve.

9 Figure 3 shows a cross sectional view of an additional embodiment of the invention in a
10 position with a closed valve.

11 Figure 4 shows the same embodiment as in Figure 3 in a position with an opened valve.

12 Figure 5 shows a cross-section view of an embodiment with a reversed principle of
13 action.

14 Figure 6 shows a cross-section view of a modified embodiment of the reversed operation
15 principle.
16

17 **DETAILED DESCRIPTION OF PREFERRED EMBODIMENTS**

18 As can be seen in Figure 1, a safety device 1 is provided. The safety device 1 surrounds
19 a component 2 that, in the figures, is located substantially in the center of the image and arranged

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1 vertically. The component 2 is designed as a fixed guide in the form of a rod along which a base
2 body 3 can slide up and down in a vertical direction Z.

3 Below the base body 3, a weight body 12 is arranged, which is pressed by means of a
4 screw system and a spring element 14 in a detachable manner against the bottom side of the base
5 body 3. Together with the base body 3, the weight body 12 is driven in an up and down
6 movement by means of a lifting device that is not shown.

7 Inside the base body 3, two clamping systems 4, arranged one above the other, are
8 provided. Each clamping system 4 presents, among other features, at least one elastically
9 deformable chamber 6. In the case of elastic deformation of the chamber 6, its dimension in the
10 radial direction changes so that a force transmission element 7 is moved in the direction toward
11 the component 2 or away from the latter. Depending on the embodiment type of clamping
12 system 4, and particularly as a function of the stiffness of the chambers 6 and the manufacturing
13 tolerances, a slight shape change in the Z direction results in large forces in the radial direction,
14 which can act via the force transmission element 7 on the component 2 to prevent a relative
15 movement between the base body 3 and the component 2 by clamping.

16 The chambers 6 of the clamping system 4 receive, via a medium supply 8, an appropriate
17 fluid, which may be pressurized air or also hydraulic oil or another fluid.

18 In the area of the medium supply 8 that runs inside the base body 3, a valve 11 is
19 provided, which can be actuated by means of an actuation element 10 that is movable in the Z
20 direction. In the example of Figure 1, the actuation element 10 is constructed as part of the

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1 weight body 12. The actuation element 10 closes the medium supply 8 so that the medium can
2 enter only into the chambers 6 of the clamping system 4, where it can build up an appropriate
3 pressure.

4 The represented arrangement of the base body with the weight body 12 (position S1)
5 applied directly against the base body, thus shows the valve 11 in the closed state.

6 In Figure 2, the same arrangement can be seen after a breakdown in the pressure supply
7 of the installation. As a result of the pressure drop, the chambers undergo pressure reduction and
8 are flattened, which results in a radial clamping force directed inward against the components 2.
9 In this state, the base body 3 is clamped against the component 2, and it can no longer be shifted
10 in the Z direction relative to this component 2.

11 The weight body 12 suspended from the base body 3 has dropped against the force of the
12 spring 14 to the degree allowed by the construction, and it is maintained in this position by
13 screwing to the base body 3. It is shifted in a position S2, and thus it releases the opening of the
14 valve 11. As a result, no medium can be compressed in the chambers 6.

15 Regardless of whether pressure is applied to the medium supply 8 or not (system failure
16 or regular operation), the clamping effect now cannot be neutralized, because the medium
17 escapes through the opened valve 11. It is only when pressurized medium is made available and
18 there is an additional lifting of the weight body 12 to the lower abutment against the base body 3
19 that the valve 11 is closed again, so that pressure is applied to the chambers 6 and in the process
20 the clamping force against the components 2 can be reduced or entirely eliminated. According

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1 to the invention, this can occur only if the lifting device – not shown – for the weight body and
2 the base body works as intended and consequently an uncontrolled lowering or downfall of the
3 weight body 12 is ruled out. In Figure 3, a modified embodiment of the invention is represented.
4 Compared to the variant of Figures 1 and 2, a modified valve 11 is represented here, which,
5 besides the connection for a medium supply 8 also presents a connection for a medium recycling
6 line 8'.

7 The actuation element 10 is designed here as a pusher, which, depending on the position
8 of the weight body 12 connected to it, closes or opens the medium recycling line 8'. The valve
9 11 is designed here in such a way that the medium used cannot exit at any time into the area
10 between the base body 3 and the weight body 12. In the standard intended operation, the
11 actuation element 10 closes the medium recycling line against the unwanted drawing off of
12 medium.

13 Analogously to the above-described system breakdown, a pressure drop in the medium
14 supply again leads to a pressure reduction in the chambers 6, resulting in a clamping force being
15 applied to the component 2.

16 The weight body 12, which is again suspended against a spring 14 on the base body 3,
17 drops to an allowable extent from the position S1 into the position S2, as represented in Figure 4.
18 In the process, the actuation element 10 opens the medium recycling line 8', so that the pressure
19 that may have been reestablished in the medium supply is released through the medium
20 recycling line 8', without being applied to the chambers 6 of the clamping systems 4. In

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1 particular when hydraulic oil is used, the supplied oil is thus reliably recycled through the
2 medium recycling line 8'.

3 The pressure medium made available in this manner, however, can again apply pressure
4 to the chambers 6, and thus release the clamping against the components 2, only if the actuation
5 element 10 again closes the medium recycling line 8'. For this purpose, the lifting device – not
6 shown – must again lift the weight body 12 from the position S2 into the position S1, to close the
7 valve 11 in this direction. It is only now that the packet consisting of the base body 3 and the
8 weight body 12 can again be moved as desired.

9 In Figure 5, a safety device 101 operating according to the reversed functioning principle
10 is shown. Because of symmetry, essentially only the right part of the safety device is
11 represented.

12 In that figure, a rod 102 is represented, which can be moved in the vertical direction and
13 which traverses a base body 103. The joint action of the rod 102 and of the base body 103
14 corresponds to the previous embodiments. In particular, the base body 103 again presents clamp
15 elements 104, which are suitable for applying a clamping force to the rod 102.

16 The base body 103 is attached below a fixed actuation element 110, and it is secured with
17 the latter by means of a screw-shaped abutment, which is prestressed by a spring 114. The
18 actuation element 110 here presents a part of the medium supply 108, which is guided to the
19 base body from above the base body 103. The valve function proper is ensured by the very
20 simply designed valve 111.

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1 This safety device functions as follows:

2 During normal operation (not shown here), the base body 103 abuts with its top surface
3 against the actuation element 110 so that there is a continuous enclosed connection of the
4 medium supply 108 up to the chambers 106. The chambers 106 therefore are inflated so that
5 they cannot exert any clamping action on the rod 102, so that the latter can move freely up and
6 down relative to the base body 103 and the actuation element 110, particularly in the vertical
7 direction.

8 In case of a sudden pressure drop in the medium supply 108 (this case is represented in
9 Figure 5), the clamp elements 104 extend because of the falling pressure, resulting in a clamping
10 action with the rod 102. Due to the operational failure, the rod 102 (optionally with weights or
11 other components attached to it) moves downward in the vertical direction due to gravity,
12 resulting also in movement some distance downward of the base body 103 clamped to it. As a
13 result, the valve 111 opens in such a manner that medium can no longer reach the clamp
14 elements 104 from the medium supply 108 (which in addition at this time has undergone a
15 pressure reduction). The abutment represented by the screw stops the downward movement of
16 the base body 103 and the rod 102 attached to the base body so that, if there is no medium,
17 undesired movements are no longer possible.

18 At the time of a new startup of operation, or when medium is made available through the
19 medium supply 108, the clamping action of the base body 103 to the rod 102 persists as long as
20 the valve 111, or the slit located there between the base body 103 and the actuation element 110,

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1 is not closed. However, if the rod 102 is lifted so that this slit is closed, then the medium reaches
2 the clamp elements 104 again, resulting in the release of the clamping, so that the rod 102 can
3 now slide completely freely up and down.

4 The spring 114 ensures that the base body 103 does not slide downward due to its own
5 weight (that is without clamping to the rod 102), thus causing an undesired clamping.

6 It is now noted that the medium supply 108 can be led directly into the base body 103,
7 instead of through the actuation element 110. The only important factor is that, in the case of a
8 relative movement between the base body 103 and the actuation element 110, a part of the
9 medium supply line is opened to the environment, so that the pressure can escape from the clamp
10 elements 104.

11 In Figure 6, a modified form of the variant according to Figure 5 is represented. The
12 fixed actuation element 110 is here formed substantially by a pin, which extends from above into
13 a section of the medium line in the base body 103. Analogously to the mechanism of action of
14 the devices in Figures 3 and 4, the medium is led here out of a medium supply 108 into a
15 medium recycling line 108', as long as the actuation element 110 does not close this medium
16 recycling line. It is only when the base body 103 and the fixed component with the actuation
17 element 110, which is located in a fixed position above the base body, move toward each other
18 that this element closes the medium recycling line again, so that pressure can again be applied to
19 the chambers 106 by the medium applied through the medium supply 108 in such a manner that
20 a clamping against the rod 102 is released.

Exhibit B
Clean Copy of Substitute Specification
U.S. National Phase Application Serial No. 10/598,528

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1 As used herein, whether in the above description or the following claims, the terms
2 "comprising," "including," "carrying," "having," "containing," "involving," and the like are to be
3 understood to be open-ended, that is, to mean including but not limited to. Only the transitional
4 phrases "consisting of" and "consisting essentially of," respectively, shall be considered
5 exclusionary transitional phrases, as set forth, with respect to claims, in the United States Patent
6 Office Manual of Patent Examining Procedures (Eighth Edition, August 2001 as revised October
7 2005), Section 2111.03.

8 Any use of ordinal terms such as "first," "second," "third," etc., in the claims to modify a
9 claim element does not by itself connote any priority, precedence, or order of one claim element
10 over another, or the temporal order in which acts of a method are performed. Rather, unless
11 specifically stated otherwise, such ordinal terms are used merely as labels to distinguish one
12 claim element having a certain name from another element having a same name (but for use of
13 the ordinal term).

14 The above described preferred embodiments are intended to illustrate the principles of
15 the invention, but not to limit the scope of the invention. Various other embodiments and
16 modifications to these preferred embodiments may be made by those skilled in the art without
17 departing from the scope of the present invention.

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19 1000_Exhibit B_Clean Specification.wpd